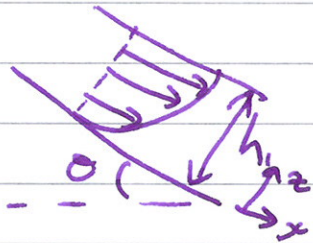


Advanced Fluid Dynamics 2012

Similar to worked example

(a)



$$\underline{u} = u(z)\underline{\hat{x}} \quad \text{so} \quad \nabla \cdot \underline{u} = 0 \text{ automatically}$$

Navier Stokes:

$$0 = -\frac{\partial p}{\partial x} + \mu \frac{\partial^2 u}{\partial z^2} + \rho g \sin \theta$$

$$0 = -\frac{\partial p}{\partial z} - \rho g \cos \theta$$

Hence pressure $p = \rho g \cos \theta (h_1 - z)$ satisfying $p(h_1) = 0$

and $\mu \frac{\partial^2 u}{\partial z^2} = -\rho g \sin \theta$

$u(0) = 0$ no no-slip
 $\frac{du}{dz}(h_1) = 0$ no shear stress

$$\Rightarrow u = \frac{\rho g \sin \theta}{2\mu} (h_1^2 - z^2)$$

4.7

Volume flux per unit width $Q = \int_0^{h_1} u dz = \frac{\rho g \sin \theta h_1^3}{3\mu}$

(b) Dissipation:

Either $D = 2\mu \int_0^{h_1} e_{ij} e_{ij} dz$ where $e_{ij} = \frac{1}{2}(\partial_i u_j + \partial_j u_i)$

Here $e_{ij} = \frac{1}{2} \begin{pmatrix} 0 & 0 & \frac{du}{dz} \\ 0 & 0 & 0 \\ \frac{du}{dz} & 0 & 0 \end{pmatrix}$ $e_{ij} e_{ij} = \frac{1}{2} \left(\frac{du}{dz} \right)^2$

$$\text{So } D = \mu \int_0^{h_1} \left(\frac{du}{dz} \right)^2 dz = \mu \int_0^{h_1} \left(\frac{\rho g \sin \theta}{\mu} \right)^2 (h_1 - z)^2 dz$$

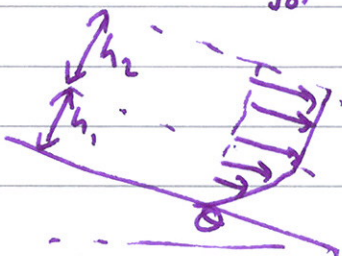
$$= \mu \left(\frac{\rho g \sin \theta}{\mu} \right)^2 \cdot \frac{h_1^3}{3} = \rho g \sin \theta Q$$

Alternatively: $0 = \mu u \frac{\partial^2 u}{\partial z^2} + \rho g \sin \theta u$

Or $0 = \mu \int_0^{h_1} u \frac{\partial^2 u}{\partial z^2} dz + \rho g \sin \theta Q$

$$\Rightarrow \int_0^{h_1} \mu \left(\frac{\partial u}{\partial z} \right)^2 dz = \rho g \sin \theta Q$$

(c)



Boundary conditions

$u = 0$ at $z = 0$ (no-slip)
 $\frac{du}{dz} = 0$ at $z = h_1 + h_2$ (no-stress)

unseen, but straightforward

4.5

$$u(z=h_1^-) = u(z=h_1^+) \quad \text{continuity of velocity at } z=h_1$$

$$\mu \frac{\partial u}{\partial z}(z=h_1^-) = \rho \nu \frac{\partial u}{\partial z}(z=h_1^+) \quad \text{continuity of shear stress at } z=h_1$$

$$h_0 < z < h_1, \quad 0 = \frac{\rho \sin \theta}{2} + \frac{\partial^2 u_1}{\partial z^2}$$

$$h_1 < z < h_1 + h_2, \quad 0 = \frac{\rho \sin \theta}{\rho \nu} + \frac{\partial^2 u_2}{\partial z^2}$$

$$\text{So } u_1 = \frac{\rho \sin \theta}{2} (Az - \frac{1}{2}z^2) \text{ satisfying } u_1(0) = 0$$

$$u_2 = \frac{\rho \sin \theta}{\rho \nu} (B + (h_1 + h_2)z - \frac{1}{2}z^2) \text{ satisfying } \frac{\partial u_2}{\partial z} = 0 \text{ at } z=h_1+h_2$$

$$\text{Continuity of } u: \quad \frac{\rho \sin \theta}{2} (Ah_1 - \frac{h_1^2}{2}) = \frac{\rho \sin \theta}{\rho \nu} (B + \frac{h_1^2}{2} + h_1 h_2)$$

$$\text{Continuity of stress: } \rho \sin \theta (A - h_1) = \rho \sin \theta (h_1 + h_2 - h_1)$$

$$\Rightarrow A = h_1 + h_2 \text{ and } B = \nu (\frac{h_1^2}{2} + h_1 h_2) - \frac{h_1^2}{2} - h_1 h_2$$

$$\text{Flux } q_1 = \int_0^{h_1} u_1 dz = \frac{\rho \sin \theta}{2} [A \frac{z^2}{2} - \frac{z^3}{3}]_0^{h_1} = \frac{\rho \sin \theta}{2} (\frac{h_1^3}{3} + \frac{h_1^2 h_2}{2})$$

$$\text{Flux } q_2 = \int_{h_1}^{h_1+h_2} u_2 dz = \frac{\rho \sin \theta}{\rho \nu} [Bz + (h_1 + h_2) \frac{z^2}{2} - \frac{z^3}{6}]_{h_1}^{h_1+h_2}$$

$$= \frac{\rho \sin \theta}{\rho \nu} [(A-1)(\frac{h_1^3}{2} + h_1 h_2) h_2 + (h_1 + h_2)(\frac{(h_1 + h_2)^2}{2} - h_1^2) - \frac{(h_1 + h_2)^3}{6} + \frac{h_1^3}{6}]$$

$$\text{Total volume flux } Q = q_1 + q_2 = \frac{\rho \sin \theta}{3\nu} h_1^3 (1 + \frac{3}{2} \frac{h_2}{h_1} + \frac{3h_2}{h_1} + 3 \frac{h_2^2}{h_1^2} + \frac{1}{\rho} (\frac{h_2}{h_1})^3)$$

$$= \frac{\rho \sin \theta}{3\nu} h_1^3 (1 + 3 \frac{h_2}{h_1} + 3 (\frac{h_2}{h_1})^2 + \frac{1}{\rho} (\frac{h_2}{h_1})^3)$$

2. The interfacial ^{shear} stress is $\rho g \sin \theta h_2$. Hence the bottom layer is driven by this + down slope gravitational acceleration and so is not affected by viscosity of overlying layer.

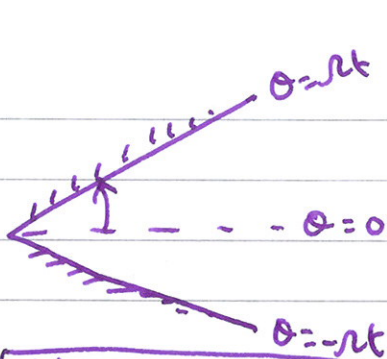
$$\text{(d) Dissipation} = \int_0^{h_1+h_2} \rho g \sin \theta u dz = \rho g \sin \theta Q$$

$$\text{[Alternative: } \int_0^{h_1} \mu (\frac{\partial u_1}{\partial z})^2 dz + \int_{h_1}^{h_1+h_2} \rho \nu (\frac{\partial u_2}{\partial z})^2 dz = \rho g \sin \theta Q.]$$

omit

similar to worksheet

2.



(polar)

$$\underline{u} = u_r \hat{r} + u_\theta \hat{\theta}$$

(b) Boundary conditions on plates:

No tangential velocity $u_r = 0$

Match normal velocity $u_\theta = \pm r\Omega$
on $\theta = \pm \alpha t$.

Sketch (a) & (b)

(a) Governing equations (negligible inertia) $\nabla \cdot \underline{u} = 0$ $0 = -\nabla p + \mu \nabla^2 \underline{u}$

Take curl of momentum: $0 = \nabla_\perp (\mu \nabla^2 \underline{u}) = \mu \nabla^2 (\nabla_\perp \underline{u})$

For 2-D motion $\underline{u} = \nabla_\perp \psi \hat{z}$ $\hat{z} = \text{unit vector } \perp \text{ to motion.}$

$$\begin{aligned} \text{Then } (\nabla_\perp \underline{u})_i &= (\nabla_\perp (\nabla_\perp \psi \hat{z}))_i = \epsilon_{ijk} \frac{\partial}{\partial x_j} \epsilon_{klp} \frac{\partial}{\partial x_p} \psi \delta_{iz} \\ &= (\delta_{ip} \delta_{jq} - \delta_{iq} \delta_{jp}) \frac{\partial}{\partial x_j} \frac{\partial}{\partial x_p} \psi \delta_{iz} \\ &= \frac{\partial}{\partial x_j} \frac{\partial \psi}{\partial x_i} - \nabla^2 \psi \delta_{iz} = -\nabla^2 \psi \hat{z} \end{aligned}$$

Thus $\nabla^2 \nabla_\perp \underline{u} = 0 \Rightarrow \nabla^4 \psi = 0$. Also note $\nabla \cdot \underline{u} = 0$.

(c) In terms of streamfunction, $u_r = \frac{1}{r} \frac{\partial \psi}{\partial \theta}$ $u_\theta = -\frac{\partial \psi}{\partial r}$

Boundary conditions demand $\frac{\partial \psi}{\partial \theta} = 0$ $\frac{\partial \psi}{\partial r} = \mp r\Omega$ on $\theta = \pm \alpha t$.

So seek solution proportional to r^2 : $\psi = f(\theta) r^2$.

$$\nabla^2 \psi = 2f + 2f'' \quad \nabla^4 \psi = \frac{1}{r^2} (f'''' + 4f'') = 0$$

Hence $f = A \cos 2\theta + B \sin 2\theta + C\theta + D$. A, B, C, D constants

$$\psi = r^2 (A \cos 2\theta + B \sin 2\theta + C\theta + D)$$

$$\frac{\partial \psi}{\partial r} = 2r (A \cos 2\theta + B \sin 2\theta + C\theta + D) = \mp r\Omega \quad \text{on } \theta = \pm \alpha t$$

$$\Rightarrow A \cos 2\alpha t + B \sin 2\alpha t + C\alpha t + D = -\frac{\Omega}{2\alpha}$$

$$\text{and } A \cos 2\alpha t - B \sin 2\alpha t - C\alpha t + D = \frac{\Omega}{2\alpha}$$

14 lectures

6

Method in lectures - different BUs here.

$$\frac{\partial \psi}{\partial \theta} = r^2 (-2A \sin 2\theta + 2B \cos 2\theta + C) = 0 \quad \theta = \pm \pi/4$$

$$\Rightarrow -2A \sin 2\pi/4 + 2B \cos 2\pi/4 + C = 0$$

$$\text{and } 2A \sin 2\pi/4 + 2B \cos 2\pi/4 + C = 0$$

Here $A=0, D=0$ and $B \sin 2\pi/4 + C \pi/4 = -\frac{\Omega}{2}$
 $2B \cos 2\pi/4 + C = 0$

$$\Rightarrow B (\sin 2\pi/4 - 2\pi/4 \cos 2\pi/4) = -\frac{\Omega}{2}$$

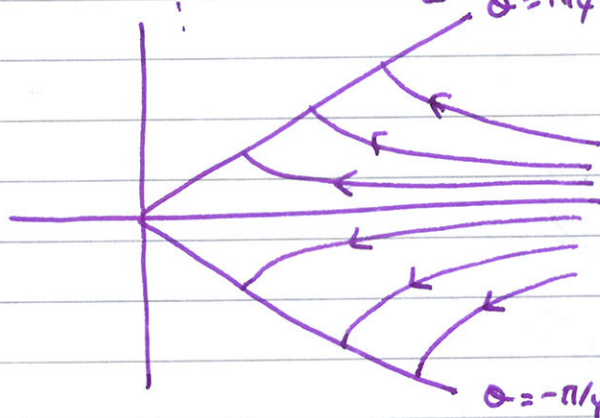
$$\Rightarrow B = \frac{-\Omega}{2(\sin 2\pi/4 - 2\pi/4 \cos 2\pi/4)}$$

$$\text{and } C = \frac{+2\Omega \cos 2\pi/4}{2(\sin 2\pi/4 - 2\pi/4 \cos 2\pi/4)}$$

Thus streamfunction $\psi = -\frac{1}{2} \Omega r^2 \left(\frac{\sin 2\theta - 2\theta \cos 2\theta}{\sin 2\pi/4 - 2\pi/4 \cos 2\pi/4} \right)$

8

When $\pi/4 = \pi/4$ $\psi = -\frac{1}{2} \Omega r^2 \sin 2\theta$



Streamlines

$$\psi = \text{const}$$

$$\Rightarrow r \cos \theta \cdot r \sin \theta = \text{const}$$

$$\Rightarrow xy = \text{const.}$$

2

(d) Navier Stokes equations: $\frac{\partial \underline{u}}{\partial t} + \underline{u} \cdot \nabla \underline{u} = -\frac{1}{\rho} \nabla p + \nu \nabla^2 \underline{u}$

$$\underline{u} \cdot \nabla \underline{u} \sim \left(\frac{\Omega r}{r} \right)^2 \quad \nu \nabla^2 \underline{u} \sim \nu \frac{\Omega}{r^2}$$

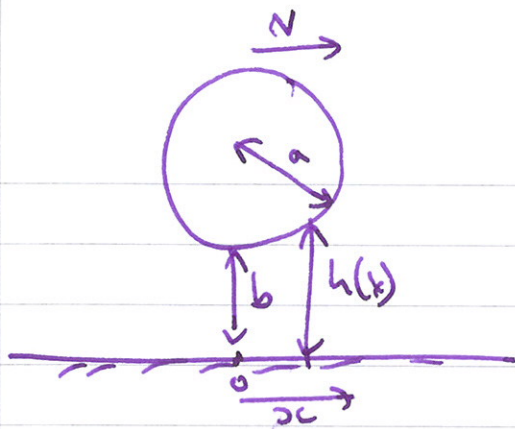
Inertial terms become comparable with viscous when $\Omega^2 r \sim \nu \frac{\Omega}{r}$
 $\Rightarrow r^2 \sim \frac{\nu}{\Omega}$ i.e. $r \sim \left(\frac{\nu}{\Omega} \right)^{1/2}$

The viscous solution is valid for $r << \left(\frac{\nu}{\Omega} \right)^{1/2}$

under

6

3.
Similar to
lectures



$$(a) \quad x^2 + (a - (h-b))^2 = a^2$$

$$\Rightarrow h = a + b - (a^2 - x^2)^{1/2}$$

$$= a + b - a \left(1 - \frac{x^2}{a^2}\right)^{1/2}$$

$$\approx b + \frac{1}{2} \frac{x^2}{a} + \dots$$

(b) Lubrication regime ($b/a \ll 1$)

$$0 = -\frac{\partial p}{\partial x} + \mu \frac{\partial^2 u}{\partial z^2} \quad \text{subject to} \quad u(0) = -V$$

$$u(h) = 0$$

$$\Rightarrow u = \frac{1}{2\mu} \frac{\partial p}{\partial x} z^2 + Az + B$$

$$u(0) = -V \Rightarrow B = -V, \quad u(h) = 0 = \frac{1}{2\mu} \frac{\partial p}{\partial x} h^2 + Ah - V$$

$$\Rightarrow A = -\frac{1}{2\mu} \frac{\partial p}{\partial x} h + \frac{V}{h}$$

$$\Rightarrow u = \frac{1}{2\mu} \frac{\partial p}{\partial x} (z^2 - hz) + \frac{Vz}{h} - V$$

$$\text{Volume flux } q = \int_0^h u \, dz = -\frac{h^3}{12\mu} \frac{\partial p}{\partial x} - \frac{Vh}{2}$$

$$\text{But } q = \text{constant} \Rightarrow \frac{\partial p}{\partial x} = -\frac{12\mu}{h^3} \left(q + \frac{Vh}{2}\right)$$

$$\text{There is no imposed pressure gradient so } \int_{-a}^a \frac{\partial p}{\partial x} \, dx = 0$$

$$\Rightarrow q \int_{-a}^a \frac{1}{h^3} \, dx = -\frac{V}{2} \int_{-a}^a \frac{1}{h^2} \, dx$$

$$\int_{-a}^a \frac{1}{h^3} \, dx = \int_{-a}^a \frac{1}{b^3} \sqrt{2ab} \frac{1}{(1+s^2)^{3/2}} \, ds = \frac{\sqrt{2ab}}{b^3} \cdot \frac{3\pi}{8}$$

$$\int_{-a}^a \frac{1}{h^2} \, dx = \frac{1}{b^2} \sqrt{2ab} \int_{-a}^a \frac{1}{(1+s^2)^{3/2}} \, ds = \frac{\sqrt{2ab}}{b^2} \cdot \frac{\pi}{2}$$

} from hint

$$\text{Thus } q = -\frac{V}{2} \cdot \frac{\sqrt{2ab}}{b^2} \cdot \frac{\pi}{2} \cdot \frac{8}{3\pi} \frac{b^3}{\sqrt{2ab}} = -\frac{2}{3} Vb$$

$$\text{Thus } \frac{\partial p}{\partial x} = -\frac{12\mu}{h^3} \left(-\frac{2}{3} Vb + \frac{Vh}{2}\right) = \frac{\mu V}{h^2} \left(\frac{8b}{h} - 6\right)$$

Method shown involves for other examples

Method in lectures, but this example is unusual

$$(c) \text{ shear stress on lower boundary} = \mu \frac{du}{dy} \Big|_{y=0}$$

$$= \mu \left(-\frac{h}{2\mu} \frac{dp}{dx} + \frac{V}{h} \right)$$

$$= \mu \left(-\frac{V}{2h} \left(\frac{8b}{h} - 6 \right) + \frac{V}{h} \right) = \mu V \left(\frac{4}{h} - \frac{4b}{h^2} \right)$$

$$\text{Force per unit width } F = \int_{-\infty}^{\infty} \mu \frac{du}{dy} \Big|_{y=0} dx = \mu V 4 \int_{-\infty}^{\infty} \frac{1}{h} - \frac{b}{h^2} dx$$

$$\int_{-\infty}^{\infty} \frac{1}{h} dx = \frac{1}{b} \sqrt{2ab} \cdot \pi \quad [\text{from hint}]$$

$$\text{Thus } F = 4\mu V \cdot \left(\frac{\pi}{b} \sqrt{2ab} - \frac{\sqrt{2ab}}{b} \frac{\pi}{2} \right) = 2\pi \mu V \frac{\sqrt{2ab}}{b}$$

Drag on cylinder = $-F$ because in absence of media $\int_S \sigma_{ij} n_j dS = 0$
and here $S = \{\text{surface of cylinder}\} \cup \{\text{plane}\}$

Thus force F must be applied to maintain the motion.

[* Sufficient to appeal to Newton's III law]

2-D case is in lectures

$$4(a) \quad F = \int_{-\infty}^{\infty} u^2 r \, dz$$

$$\frac{dF}{dr} = \int_{-\infty}^{\infty} u^2 + 2ru \frac{du}{dr} \, dz$$

$$= \int_{-\infty}^{\infty} u^2 + 2r \left(v \frac{d^2 u}{dz^2} - w \frac{du}{dz} \right) dz$$

$$= \int_{-\infty}^{\infty} u^2 + 2r \left(v \frac{d^2 u}{dz^2} - \frac{d}{dz}(uw) + u \frac{dw}{dz} \right) dz \quad \text{but } \frac{dw}{dz} = -\frac{1}{r} \frac{d}{dr}(ru)$$

$$= \int_{-\infty}^{\infty} u^2 + 2r \left(v \frac{d^2 u}{dz^2} - \frac{d(uw)}{dz} \right) - 2ru \cdot \frac{1}{r} \frac{d}{dr}(ru) \, dz$$

$$= \int_{-\infty}^{\infty} 2r \left(v \frac{d^2 u}{dz^2} - \frac{d(uw)}{dz} \right) - \frac{dF}{dr}$$

$$\Rightarrow \frac{dF}{dr} = r \left[v \frac{du}{dz} - uw \right]_{-\infty}^{\infty} = 0 \quad \text{as } \frac{du}{dz}, u \rightarrow 0 \text{ as } |z| \rightarrow \infty$$

6 Hence $F = \text{constant}$.

(b) Scaling: $u^2 r \delta \sim F$ and from Navier-Stokes $\frac{u^2}{r} \sim \frac{\nu u}{\delta^2}$

$$\Rightarrow u \sim \frac{\nu r}{\delta^2} \quad \text{and} \quad \left(\frac{\nu r}{\delta^2} \right)^2 r \delta \sim F$$

$$\text{Hence} \quad \delta^3 \sim \frac{\nu^2}{F} r^3$$

$$\Rightarrow \delta \sim \left(\frac{\nu^2}{F} \right)^{1/3} r \quad \text{and} \quad u \sim \frac{\nu r}{\left(\frac{\nu^2}{F} \right)^{2/3} r^2} \sim \left(\frac{F}{\nu} \right)^{1/3} \frac{1}{r}$$

$$\text{In terms of streamfunction } u = -\frac{1}{r} \frac{d\Phi}{dz} \Rightarrow \Phi \sim u r \delta$$

$$\Rightarrow \Phi \sim \left(\frac{F}{\nu} \right)^{1/3} \cdot \left(\frac{\nu^2}{F} \right)^{1/3} r = (F\nu)^{1/3} r$$

6 Hence pose $\delta(r) = \left(\frac{\nu^2}{F} \right)^{1/3} r$ and $\Phi = -\left(F\nu \right)^{1/3} r f\left(\frac{z}{\delta} \right)$
 1 "Thin" implies $\delta/r \ll 1 \Rightarrow \left(\frac{\nu^2}{F} \right)^{1/3} \ll 1$. Reynolds Number $= \frac{(F\nu)^{1/3}}{\nu} = \left(\frac{F}{\nu^2} \right)^{1/3}$

(c) On substitution into momentum equation, gives $f''' + ff'' + f'^2 = 0$ thin
near
 $Re \gg 1$

Boundary condition: (i) $u \rightarrow 0$ as $z \rightarrow \infty \Rightarrow f' \rightarrow 0$ as $\frac{z}{\delta} \rightarrow \infty$

2.1

Similar to 2-D reasoning

(ii) Symmetry in $z=0$ for velocity field u

$\Rightarrow f(y)$ is odd function $[f(0)=0, f'(0)=0]$

1

$$(iii) \int_{-\delta}^{\delta} u^2 dz = \pi \Rightarrow \int_{-\delta}^{\delta} \left(\frac{1}{r} \frac{\partial \Phi}{\partial z} \right)^2 r \delta dy = \pi$$

$$\text{but } \frac{\partial \Phi}{\partial z} = - \left(\frac{r}{v} \right)^{1/3} f'$$

$$\Rightarrow \int_{-\delta}^{\delta} \left(\frac{r}{v} \right)^{2/3} \frac{1}{\delta^2} r \delta f'^2 dy = \pi$$

$$\text{but } \frac{r}{\delta} = \left(\frac{r}{v} \right)^{1/3} \Rightarrow \int_{-\delta}^{\delta} f'^2 dy = 1.$$

2

(d) Integrating $f''' + ff'' + f'^2 = 0$

$$f'' + ff' = \text{const} = 0$$

integrating $f' + \frac{1}{2}f^2 = \frac{1}{2}c^2$ (assuming $f \rightarrow c$ as $y \rightarrow \infty$)

$$\int \frac{df}{\frac{1}{2}(c^2 - f^2)} = \int dy$$

$$f = c \tanh \int \frac{2c \operatorname{sech}^2 ds}{c^2(1 - \tanh^2)} = \int dy$$

$$\Rightarrow \frac{2}{c} \tanh^{-1}\left(\frac{f}{c}\right) = y + c_1, \text{ but } c_1 = 0 \text{ as } f(y) \text{ is odd}$$

$$\Rightarrow f = c \tanh \frac{cy}{2}$$

To determine c : $\frac{df}{dy} = \frac{c^2}{2} \operatorname{sech}^2 \frac{cy}{2}$ $\frac{c^4}{4} \int_{-\infty}^{\infty} \operatorname{sech}^4 \frac{cy}{2} dy = 1$

$$p = \frac{cy}{2} \Rightarrow \frac{c^3}{2} \int_{-\infty}^{\infty} \operatorname{sech}^4 p dp = 1$$

$$\int_{-\infty}^{\infty} \operatorname{sech}^4 p dp = \int_{-\infty}^{\infty} \operatorname{sech}^2(1 - \tanh^2 p) dp = \left[\tanh p - \frac{1}{3} \tanh^3 p \right]_{-\infty}^{\infty} = \frac{4}{3}$$

$$\Rightarrow c^3 = 3/2 \quad \text{and} \quad f = \left(\frac{3}{2} \right)^{1/3} \tanh \left[\left(\frac{3}{2} \right)^{1/3} \frac{y}{2} \right]$$

$$rW = \frac{\partial \Phi}{\partial r} \rightarrow - \left(\frac{3\pi v}{2} \right)^{1/3} \text{ as } |z/\delta| \rightarrow \infty. \text{ Non zero because jet enters fluid.}$$

2

Unseen, but similar to 2-D

Unseen, but essentially identical to 2-D case.

5. The undisturbed flow $\underline{u} = U_0 \hat{x}$, $p = p_0$

(a) Introduce perturbation $\underline{u}_1 \equiv \underline{u}_1(x, z, t)$ [2D perturbation]

$$\nabla \cdot \underline{u}_1 = 0 \Rightarrow \frac{\partial u_1}{\partial x} + \frac{\partial w_1}{\partial z} = 0$$

$$\rho \frac{D\underline{u}}{Dt} = -\nabla p \Rightarrow \left. \begin{aligned} \rho \left(\frac{\partial u_1}{\partial t} + U_0 \frac{\partial u_1}{\partial x} + w_1 \frac{dU_0}{dz} \right) &= -\frac{\partial p_1}{\partial x} \\ \rho \left(\frac{\partial w_1}{\partial t} + U_0 \frac{\partial w_1}{\partial x} \right) &= -\frac{\partial p_1}{\partial z} \end{aligned} \right\} \text{Linearised equations}$$

$$\begin{aligned} -\nabla^2 p_1 &= \rho \left(\frac{\partial}{\partial t} \left(\frac{\partial u_1}{\partial x} + \frac{\partial w_1}{\partial z} \right) + U_0 \frac{\partial}{\partial x} \left(\frac{\partial u_1}{\partial x} + \frac{\partial w_1}{\partial z} \right) + \frac{\partial w_1}{\partial x} \frac{dU_0}{dz} + \frac{dU_0}{dz} \frac{\partial w_1}{\partial x} \right) \\ &= 2\rho \frac{dU_0}{dz} \frac{\partial w_1}{\partial x} \end{aligned}$$

Then
$$\begin{aligned} \rho \frac{\partial}{\partial t} \nabla^2 w_1 + \rho U_0 \frac{\partial}{\partial x} \nabla^2 w_1 + 2\rho \frac{dU_0}{dz} \frac{\partial w_1}{\partial x} + \rho \frac{d^2 U_0}{dz^2} w_1 &= -\frac{\partial}{\partial z} (\nabla^2 p_1) \\ &= +2\rho \frac{dU_0}{dz} \frac{\partial w_1}{\partial x} + 2\rho \frac{d^2 U_0}{dz^2} w_1 \end{aligned}$$

Hence
$$\left(\frac{\partial}{\partial t} + U_0 \frac{\partial}{\partial x} \right) \nabla^2 w_1 - \frac{d^2 U_0}{dz^2} w_1 = 0.$$

(b) Introduce $w_1 = (U_0 - c) \phi e^{ik(x-ct)}$
 $\frac{dw_1}{dz} = ik(U_0 - c) \phi e^{ik(x-ct)}$
 $\nabla^2 w_1 = \left[\frac{d^2}{dz^2} (U_0 - c) \phi - k^2 (U_0 - c) \phi \right] e^{ik(x-ct)}$

Thus
$$ik(U_0 - c) \left[(U_0 - c) \phi'' + 2U_0' \phi' + U_0'' \phi - k^2 (U_0 - c) \phi \right] - ik U_0'' (U_0 - c) \phi = 0$$

$$\Rightarrow (U_0 - c)^2 \phi'' + 2U_0' (U_0 - c) \phi' - k^2 (U_0 - c)^2 \phi = 0$$

$$\Rightarrow \frac{d}{dz} \left((U_0 - c)^2 \frac{d\phi}{dz} \right) - k^2 (U_0 - c)^2 \phi = 0$$

(c)
$$\int_0^d \phi^* \frac{d}{dz} \left((U_0 - c)^2 \frac{d\phi}{dz} \right) dz = \left[\phi^* (U_0 - c)^2 \frac{d\phi}{dz} \right]_0^d - \int_0^d (U_0 - c)^2 \left| \frac{d\phi}{dz} \right|^2 dz$$

but $\phi^* = 0$ at $z=0, d \Rightarrow \int_0^d (U_0 - c)^2 Q dz = 0$
 where $Q = \left| \frac{d\phi}{dz} \right|^2 + k^2 |\phi|^2$

Real part is unken

Introduce $c = c_r + i c_i$ $(u_0 - c)^2 = ((u_0 - c_r)^2 - c_i^2) - 2i c_i (u_0 - c_r)$

Taking real part: $\int_0^d ((u_0 - c_r)^2 - c_i^2) Q dz = 0$

imaginary part: $c_i \int_0^d (u_0 - c_r) Q dz = 0$

(d) $\max\{u\} = u_1$, $\min\{u\} = u_2$ then $(u_0 - u_1)(u_0 - u_2) \leq 0$

Hence $\int_0^d (u_0 - u_1)(u_0 - u_2) Q dz \leq 0$

If flow is unken then $\int_0^d (u_0 - c_r) Q dz = 0$ as $c_i > 0$

2

Expanding $(u_0 - u_1)(u_0 - u_2) = u_0^2 - u_0(u_1 + u_2) + u_1 u_2$

then $\int_0^d u_0(u_1 + u_2) Q dz = \int_0^d c_r(u_1 + u_2) Q dz$

and $\int_0^d u_0^2 Q dz = \int_0^d (c_i^2 + 2u_0 c_r - c_r^2) Q dz$
 $= \int_0^d (c_i^2 + c_r^2) Q dz$

unken 4

Thus $\int_0^d (c_r^2 + c_i^2 - (u_0 + u_2)c_r + u_1 u_2) Q dz \leq 0$

As $Q \geq 0$ this implies $c_r^2 + c_i^2 - (u_1 + u_2)c_r + u_1 u_2 \leq 0$

$\Rightarrow \left(c_r - \frac{(u_1 + u_2)}{2}\right)^2 + c_i^2 \leq \left(\frac{u_1 + u_2}{2}\right)^2 - u_1 u_2$
 $= \left(\frac{u_1 - u_2}{2}\right)^2$